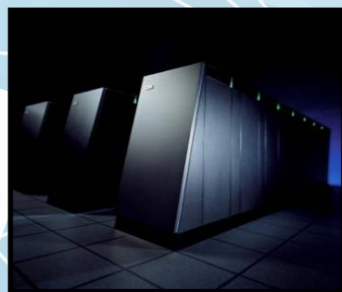
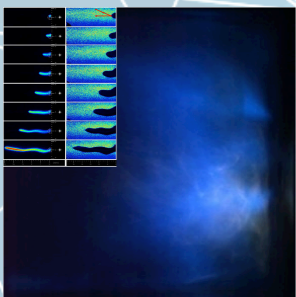
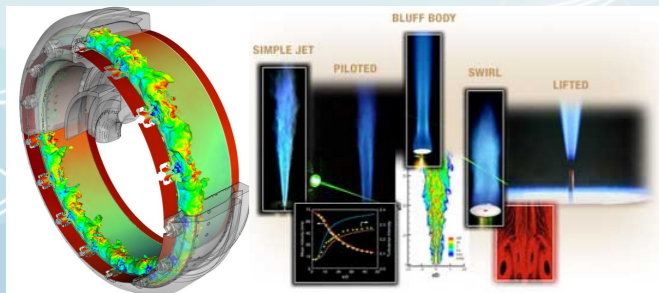




# LES of industrial turbulent reacting flows: modeling effects and challenges



**L.Y.M. Gicquel**<sup>†</sup>

B. Cuenot<sup>1</sup>, E. Riber<sup>1</sup>, G. Staffelbach<sup>1</sup>, A. Dauplain<sup>1</sup>, N. Odier<sup>1</sup>  
F. Duchaine<sup>1</sup>, O. Vermorel<sup>1</sup>, J. Dombard<sup>1</sup>, A. Misdiaris<sup>1</sup>  
T. Poinsot<sup>2</sup>

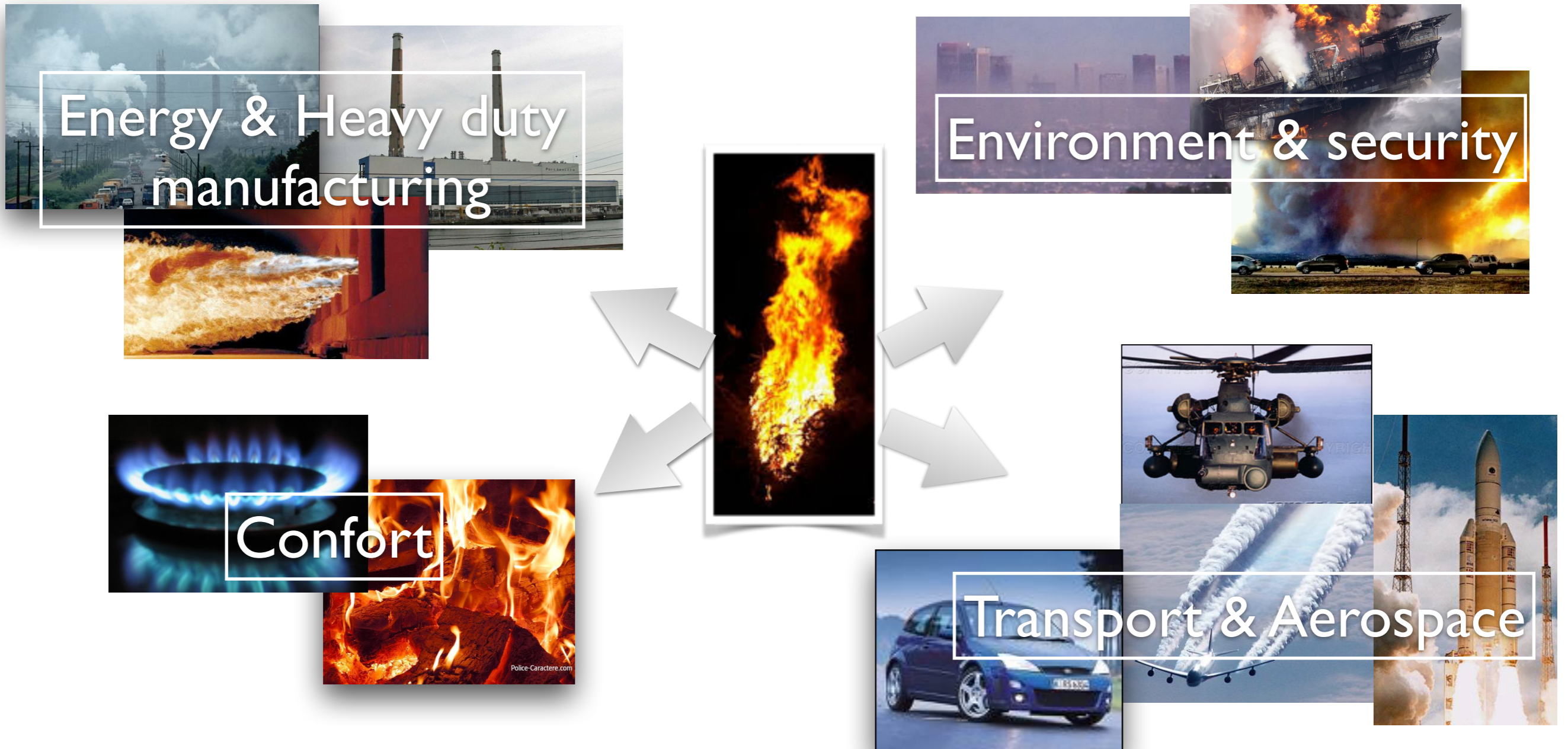
<sup>1</sup> CERFACS - CFD combustion team, Toulouse

<sup>2</sup> CNRS - IMFT, Toulouse

<sup>†</sup> <http://www.cerfacs.fr/~lgicquel>

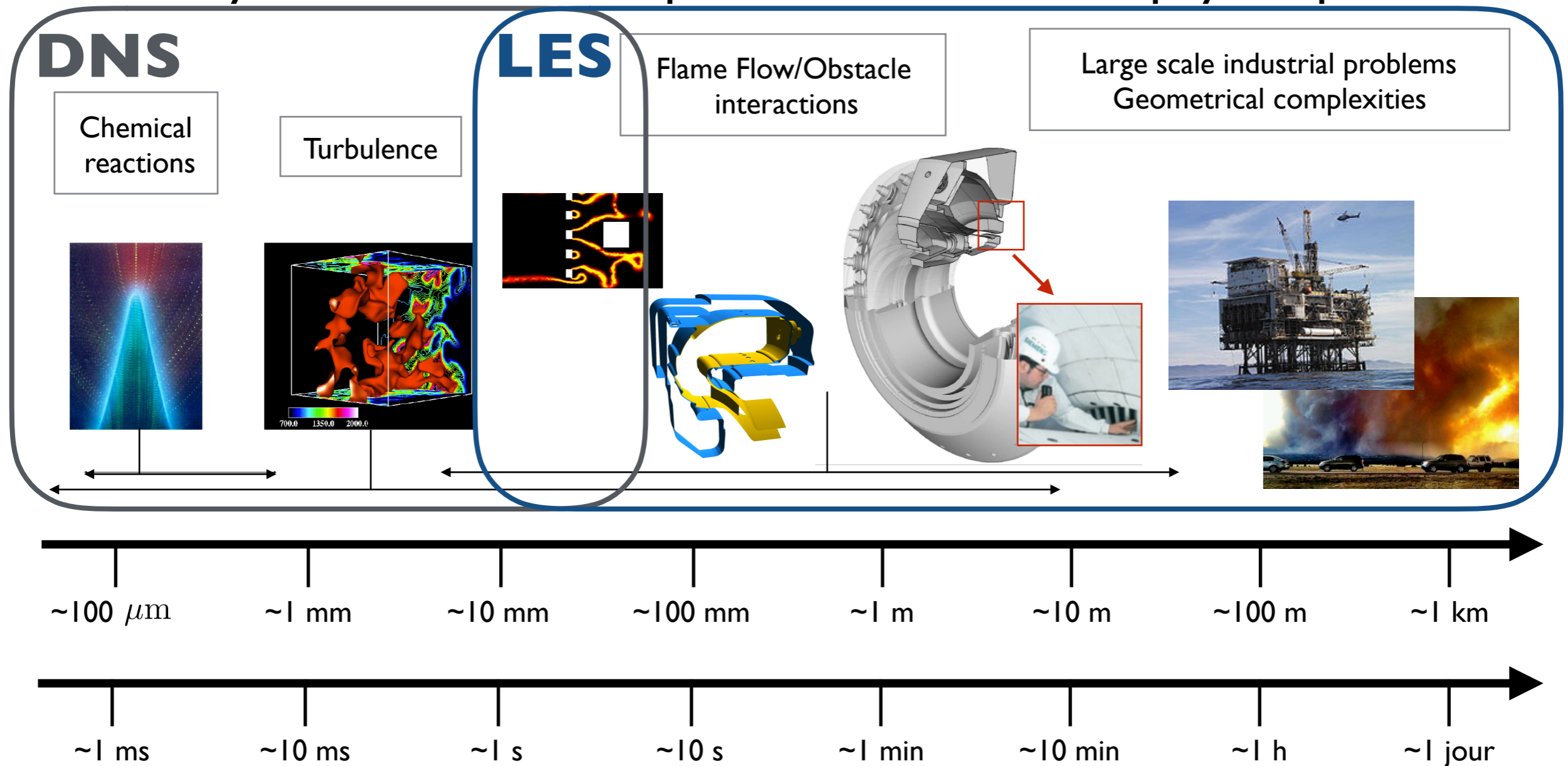
# Context and Objectives of the CFD combustion team @ CERFACS

**Combustion:** An engineering science at the cross-road between *chemistry & fluid mechanics* with strong *technological / industrial and societal* implications



# I ] HPC & turbulent reacting flow CFD

Turbulent reacting flows have been from the beginning studied and theoretically addressed as true/pure multi-scale multi-physics problems:

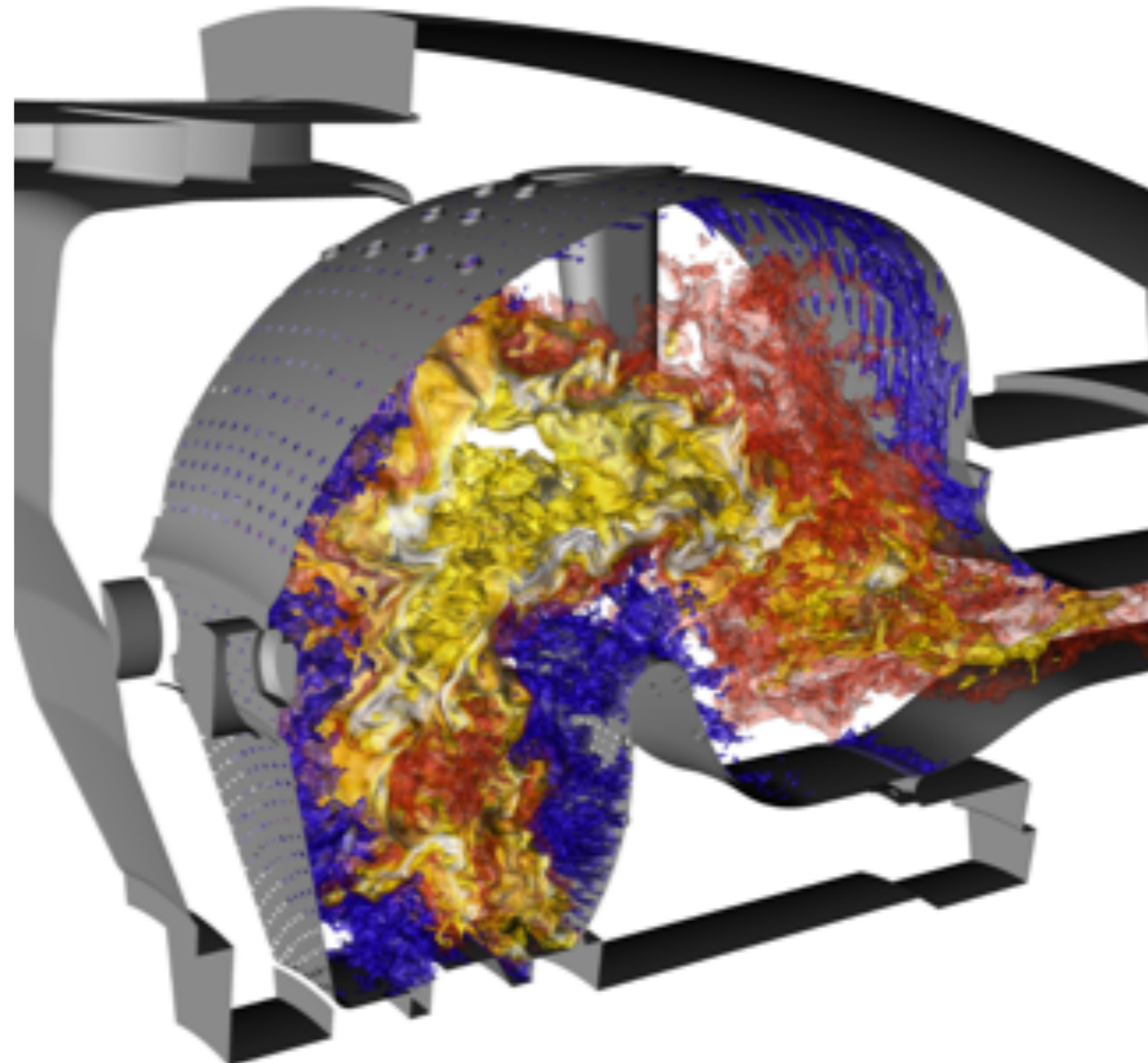
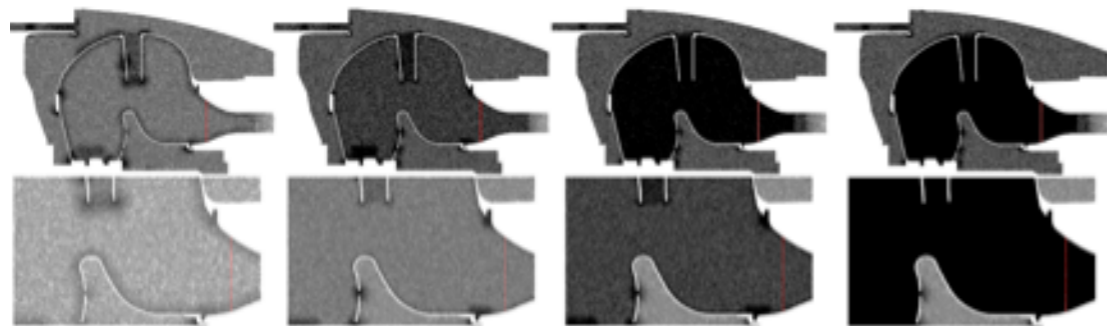


# Recent industrial achievement by SAFRAN SHE

*J. Lamouroux et al* (SAFRAN HE) - presented at the ASME TurboExpo conference, Charlotte june 2017.

- Industrial burner with **I.I.B** elements for the geometry
- Turbulent compressible, gaseous reacting LES

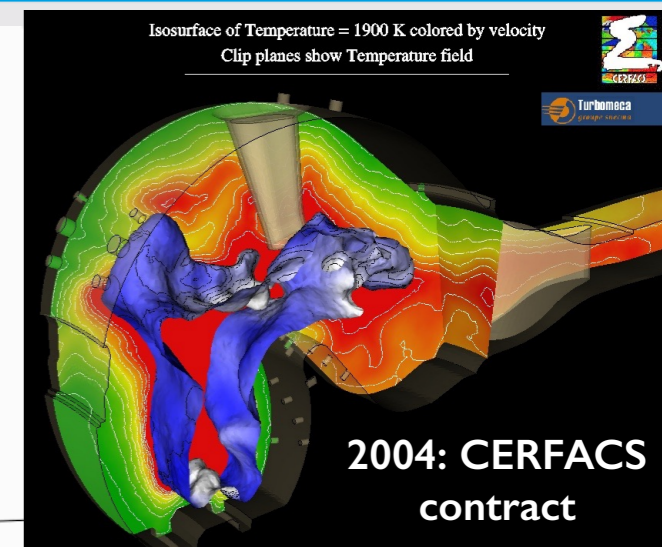
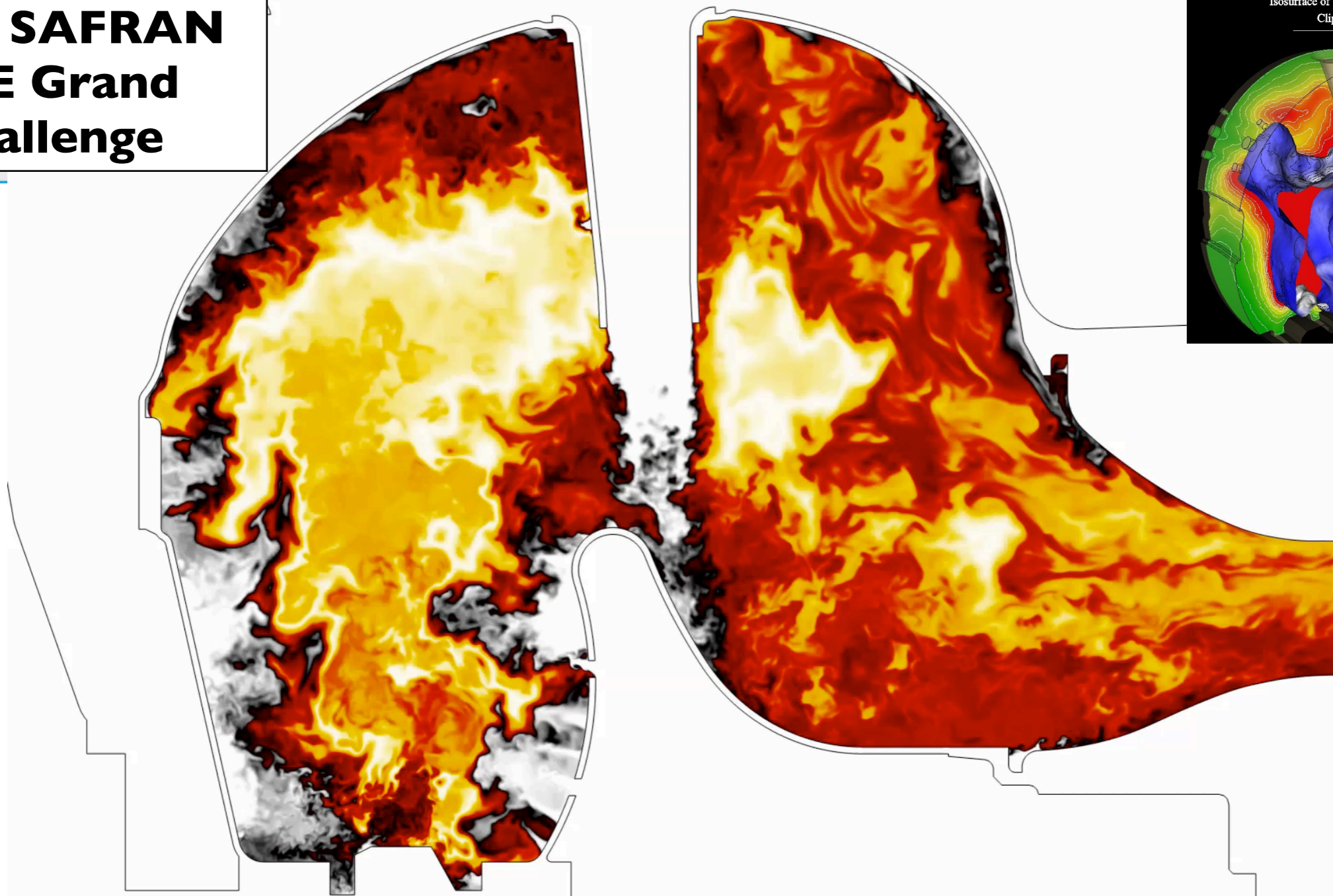
Selective refinement (chamber only) based on the **Pampa** Library using MMG3D (collaboration with *C. Lachat & C. Dobrzynski, A. Froelhy* from INRIA)



Case	# of cells	360 equiv # of cells	$\Delta_{cell}$	$\mathcal{F}_{max}$
mesh 1	11M	220M	$\Delta_0$	100
mesh 2	33M	660M	$\Delta_0/2$	50
mesh 3	220M	4400M	$\Delta_0/4$	25
mesh 4	1030M	20600M	$\Delta_0/7$	14

**CCRT supercomputing center:  
COBALT grand challenges**

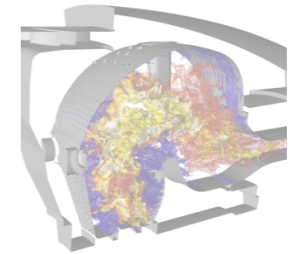
# 2016: SAFRAN SHE Grand Challenge



## Twelve years to do:

- ~1 500 times on the number of cells and ~250 times on the number of procs
- improved reduced chemistry model *PLUS* NOx and CO (crude models)
- homogeneous vs heterogeneous multi perforated plate model
- full transfer to the industry

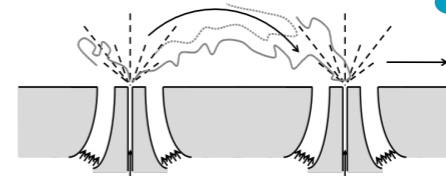
## I ] HPC & turbulent reacting flow CFD



## II ] Ignition / transient turbulent reacting flows

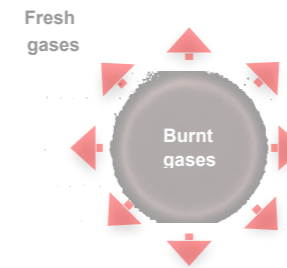
2.1 GT context: engine ignition prediction

2.2 Explosion: deflagrating fronts

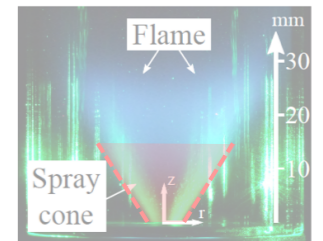


Buncefield, 2005

## III ] Difficulty of the initial phase



## IV ] GT applications: emission predictions & multi phase flows



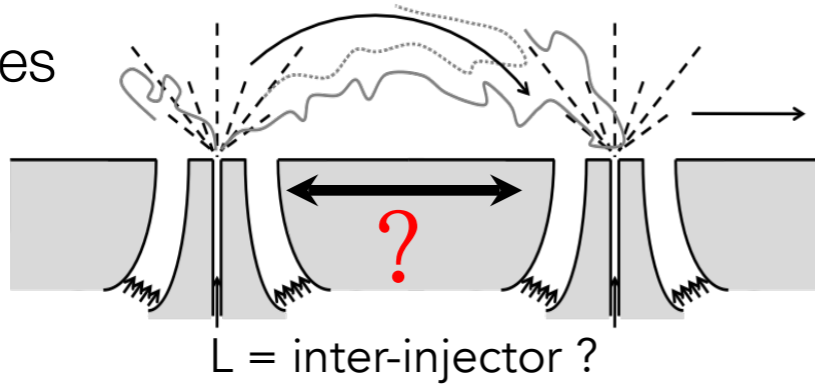
## Conclusions and perspectives

# II ] Ignition / transient turbulent reacting flows

Ignition = fully transient laminar/turbulent reacting flow

## Aeronautical GT's:

- Ignition = first design phase  
=> light around time: fctt of the burner size...
- Safety issues



## Fuel plants:

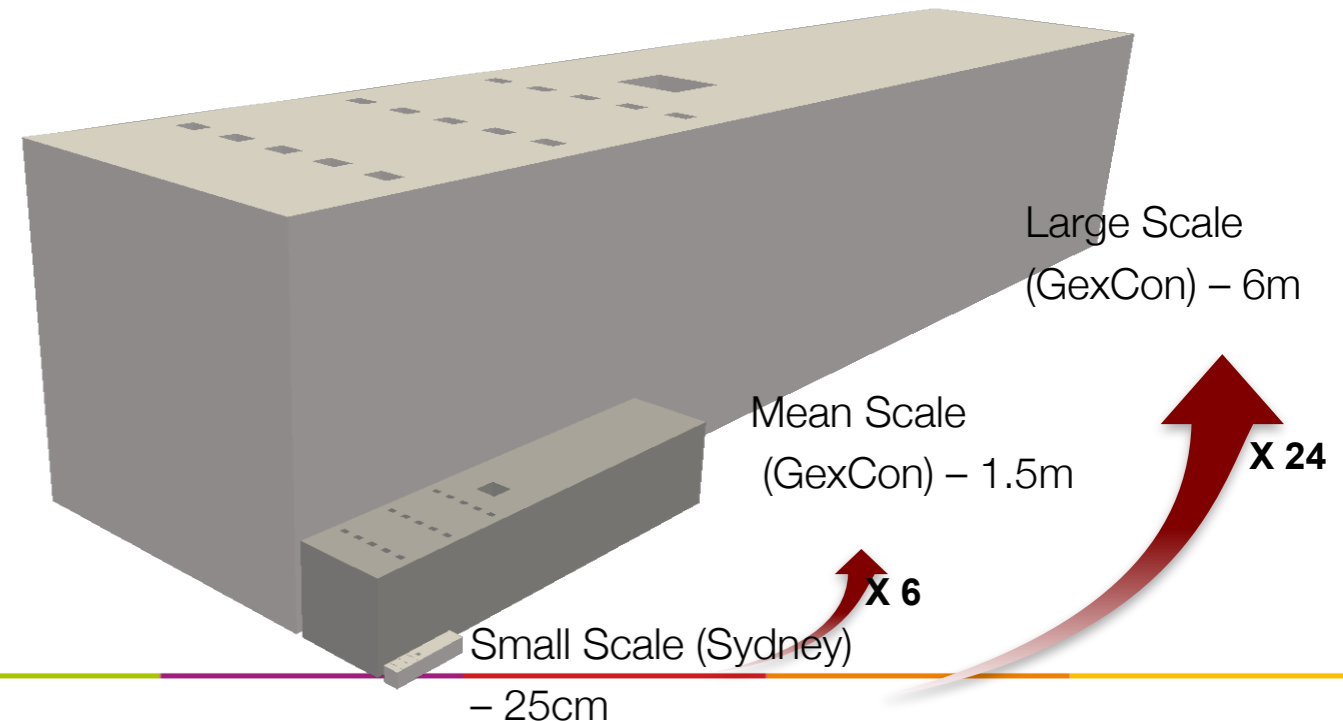
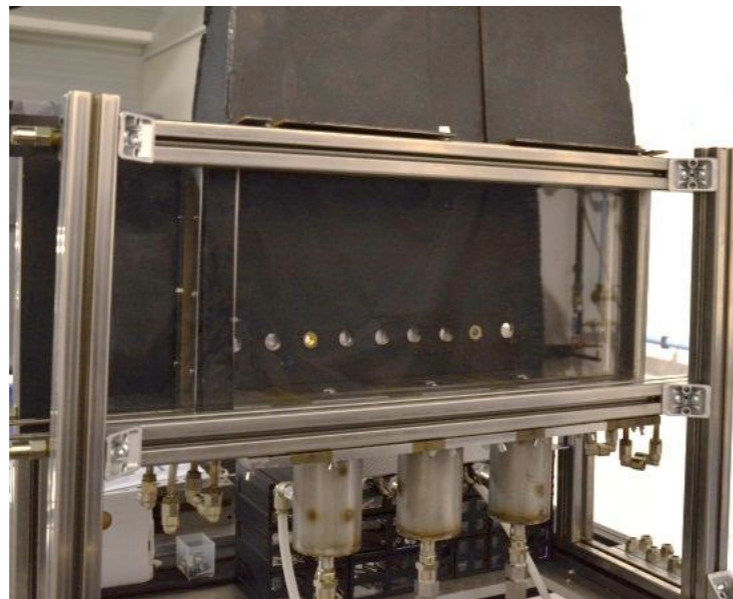
- Security issue / risk management



Deepwater horizon, 2010

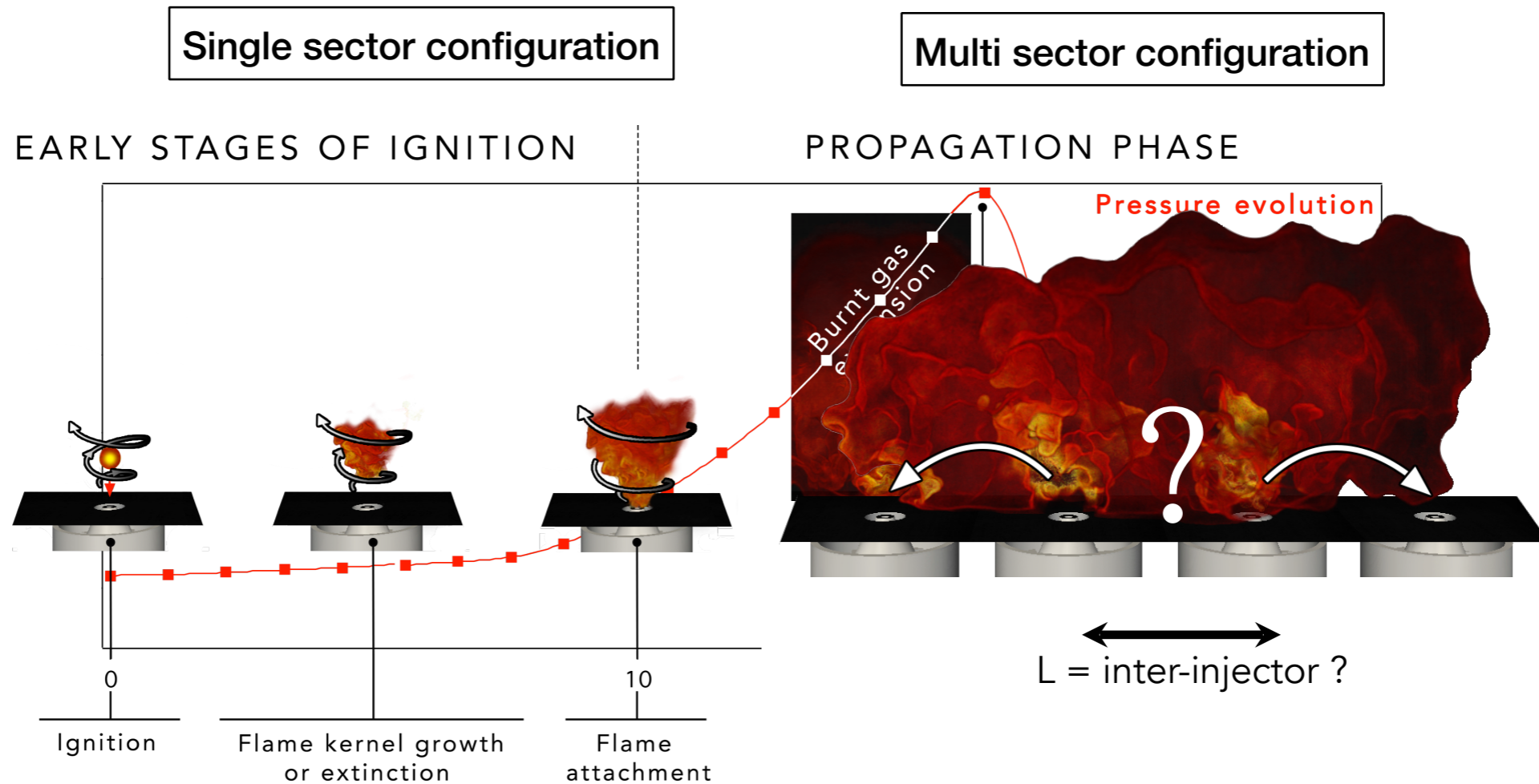


Buncefield, 2005

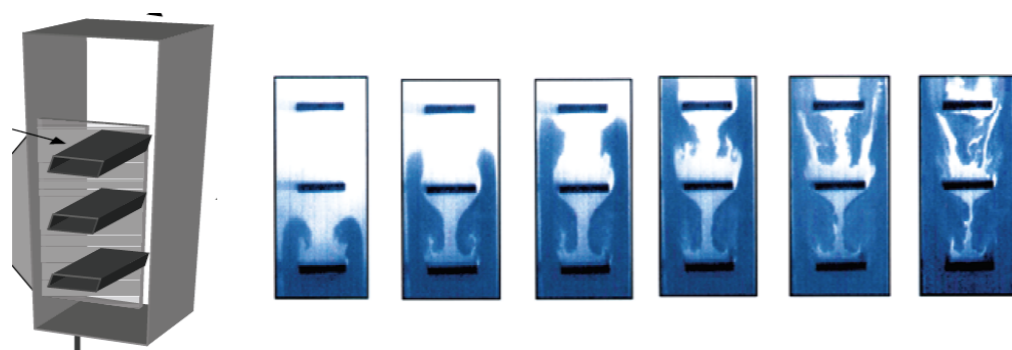


# Common features & differences

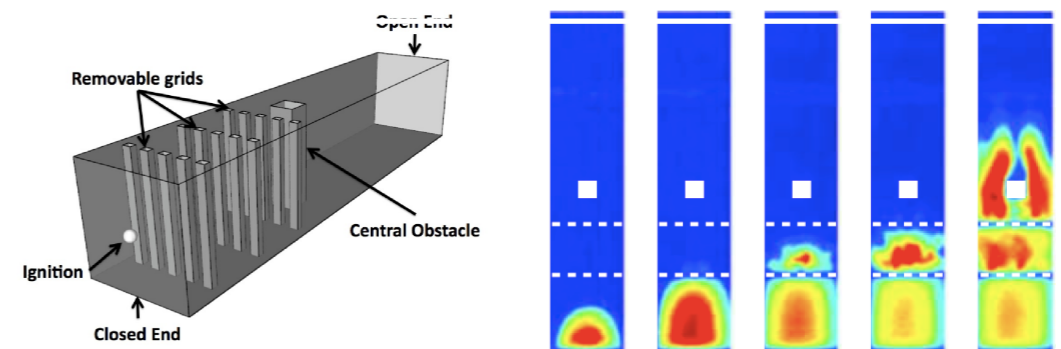
GT context



Security



Patel, S. et al. Proc. Combust. Inst. (2002)



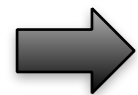
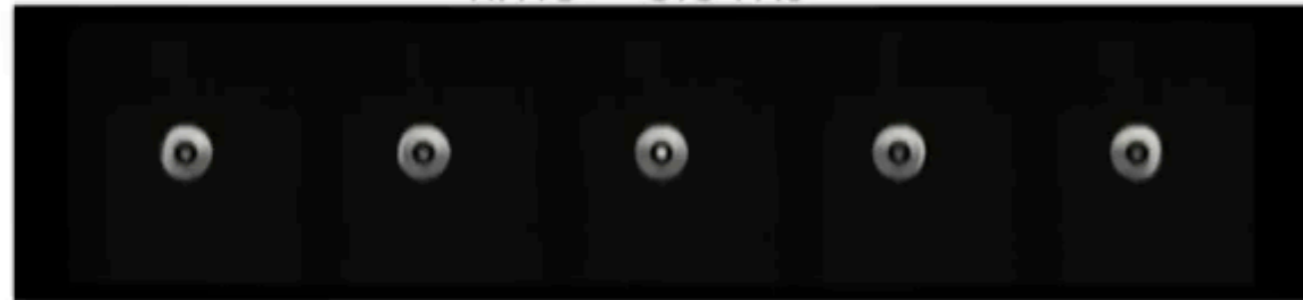
Kent, J. et al. 5th Asia-Pacific Conf. Combust., Adelaide, Australia (2005)



# 11.1 ] Light around phase - turbulent combustion dominated problem

- Low injector spacing (SP9)

- High injector spacing (SP26)



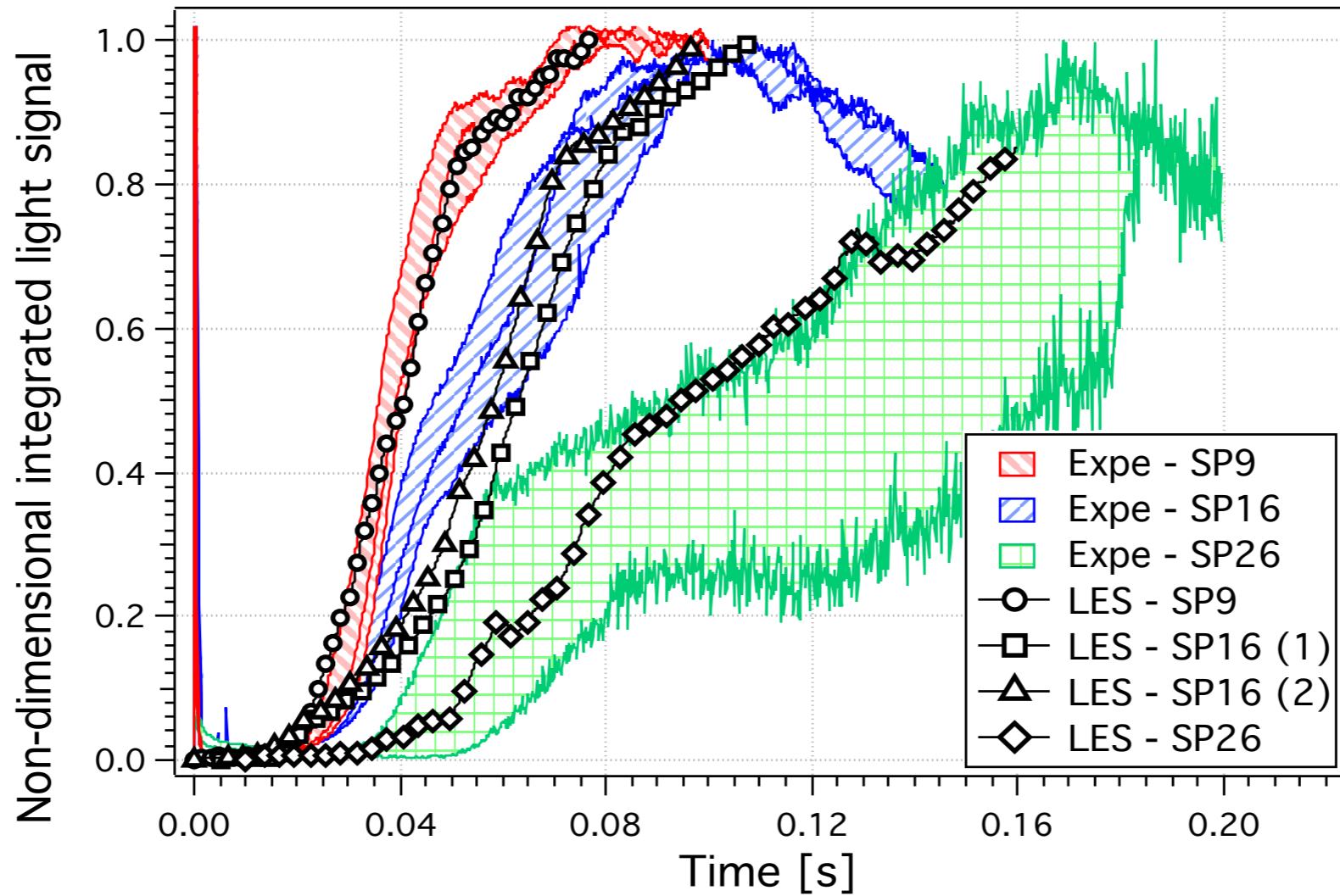
Radial flame propagation



Axial flame propagation



- Evolution of the luminous signal (CH emissions vs. Heat release images):



● Large spacing:

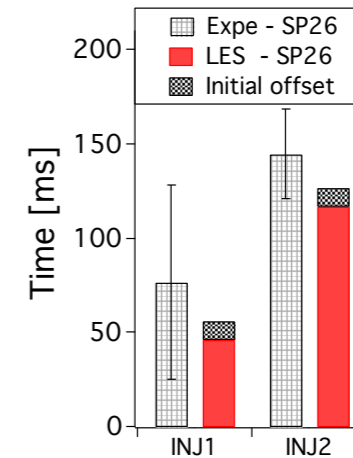
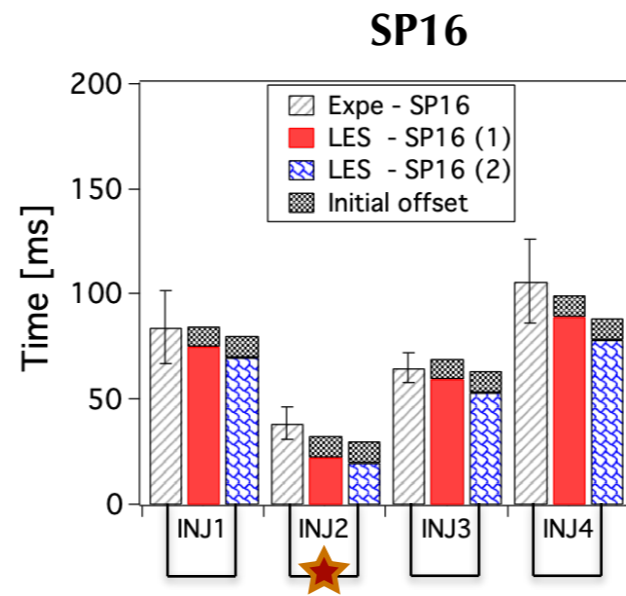
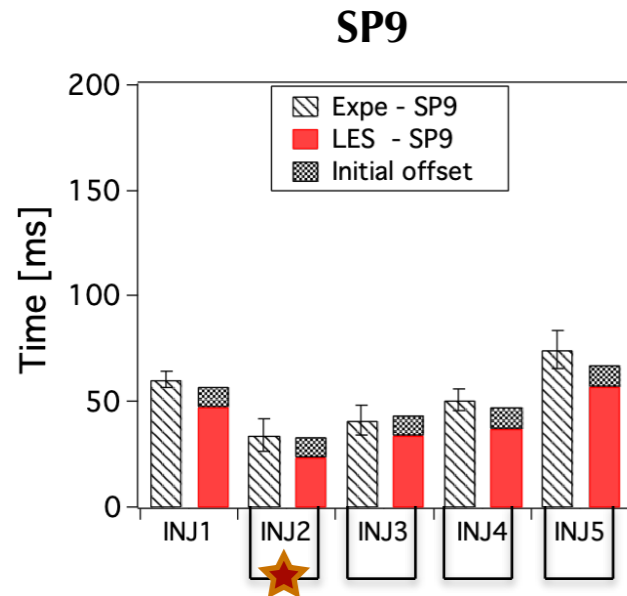
- ➔ Experimental variability
- ➔ Large overall ignition time

● Low spacing:

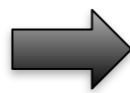
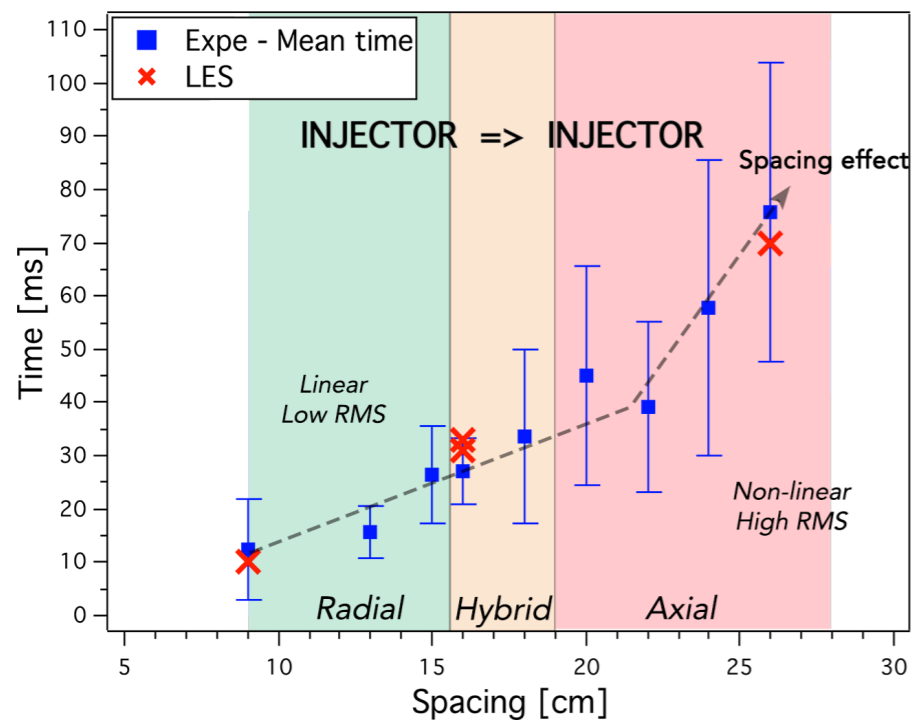
- ➔ High repeatability
- ➔ Rapid ignition process



- Ignition times for each injector



- Ignition time between two consecutive injectors



- Good estimation of the ignition times for each injectors
- 2 distinct propagation modes (inj/inj propagation times)

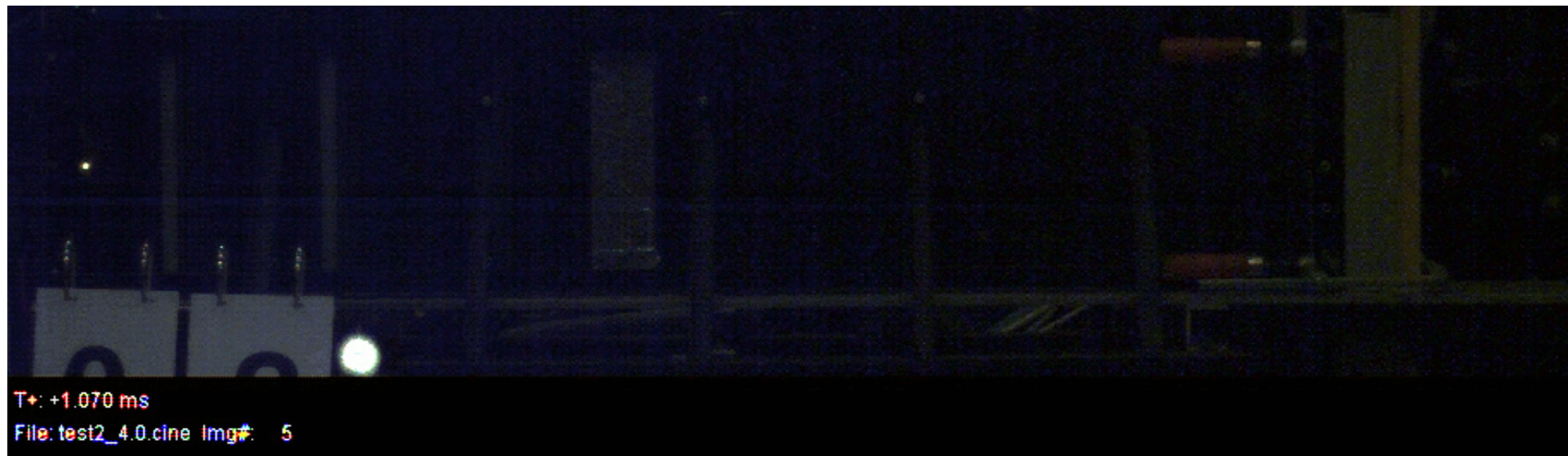
# II.2 ] Deflagration problem - fully transient and transitioning problem

Experiments

Conducted by

**cmr** Gexcon

Medium scale problem: i.e L ~1.5 m

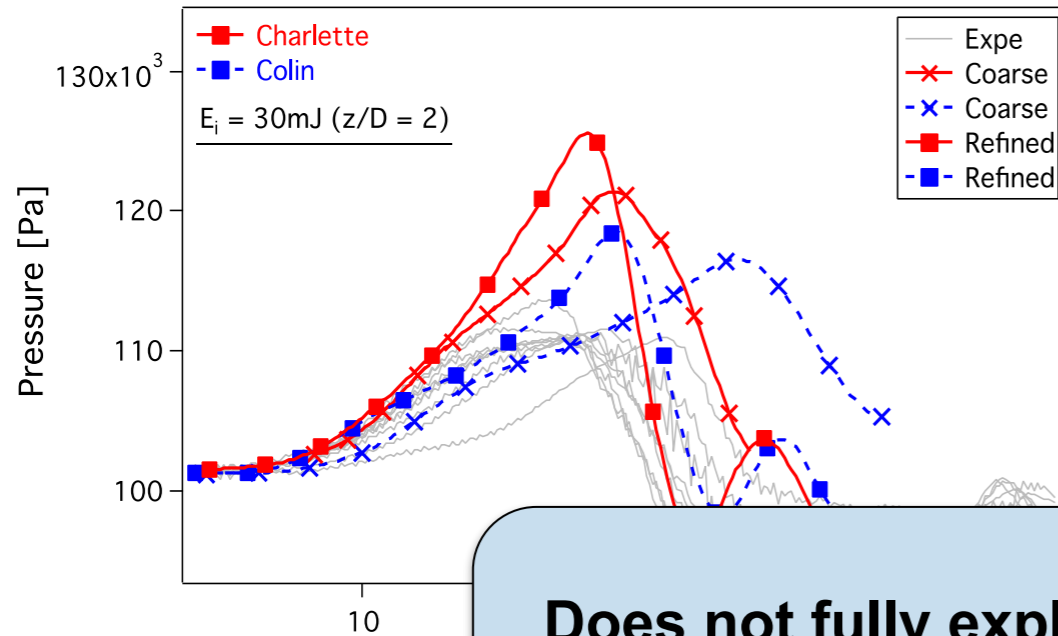


LES



Time: 1.0

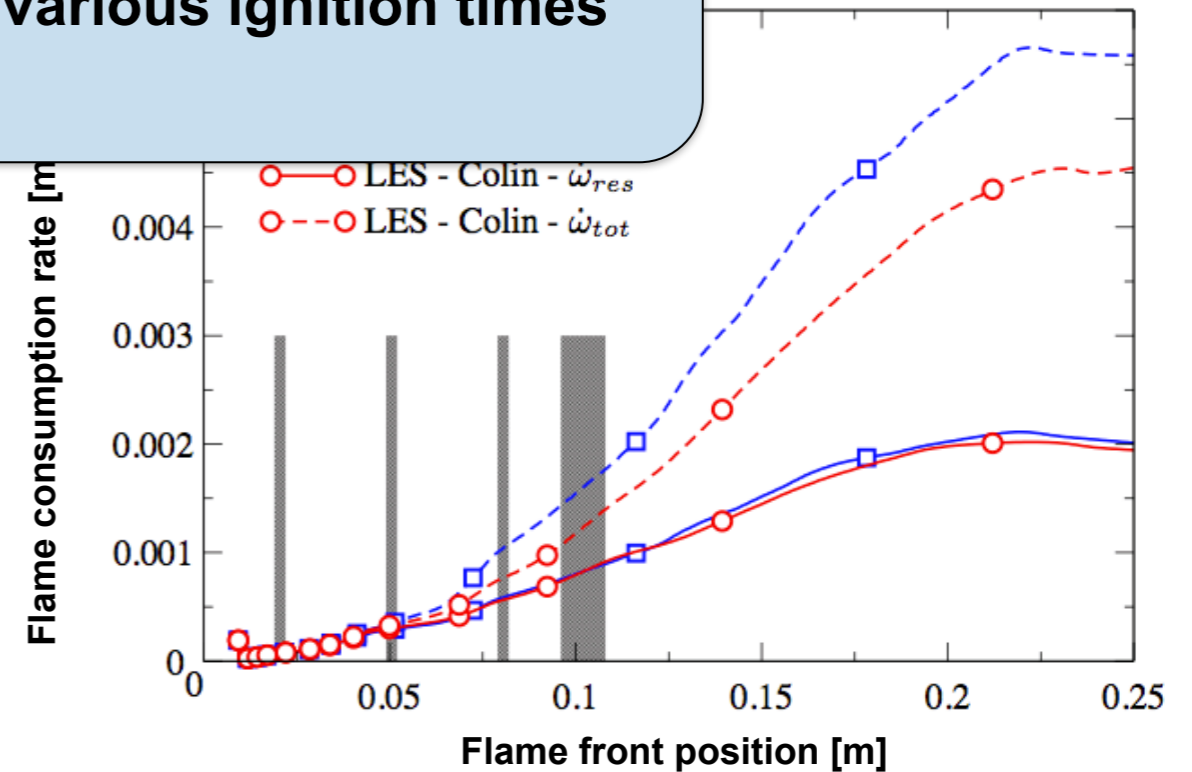
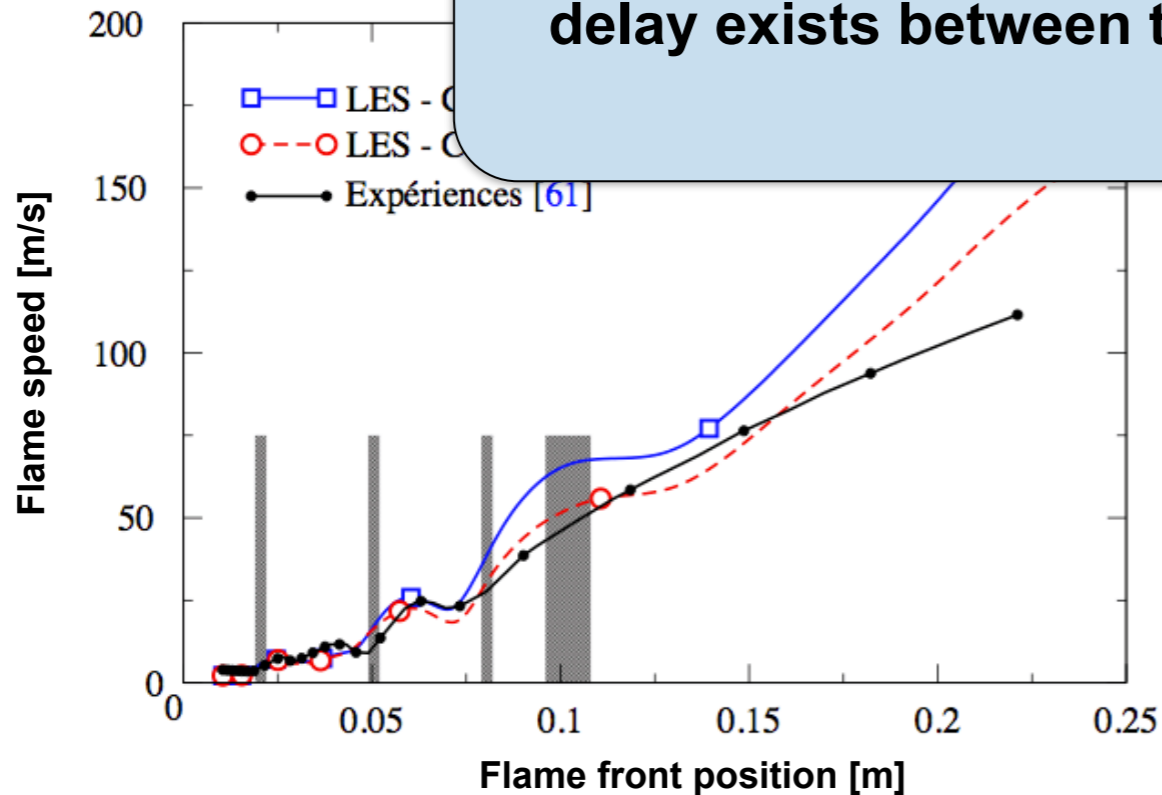
# SGS modeling impact on the predictions



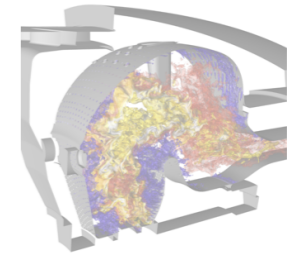
## Reaction rate:

- Fixed resolved contribution regardless of the modeling
  - Modeling effectively scales the SGS contribution and thereof the net consumption rate (=> shift in time)
  - Different modeling = different turbulent combustion flame speeds
  - Faster combustion = higher peak pressure
- ⇒ High sensitivity to SGS model !!

Does not fully explain the reason why a constant delay exists between the various ignition times



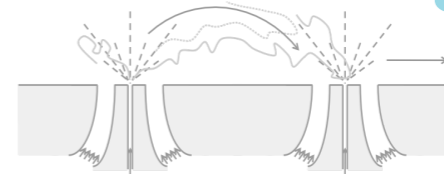
## I ] HPC & turbulent reacting flow CFD



## II ] Ignition / transient turbulent reacting flows

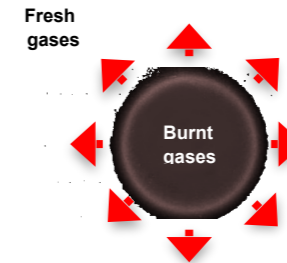
2.1 GT context: engine ignition prediction

2.2 Explosion: deflagrating fronts

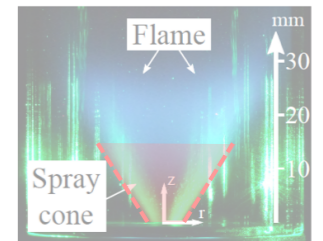


Buncefield, 2005

## III ] Difficulty of the initial phase



## IV ] GT applications: emission predictions & multi phase flows



## Conclusions and perspectives

# III ] Initial phase issues

## Major issues:

- it is not clear what is the initial state of these flow problems
- we do not know what and how much energy is deposited...
  - => pseudo deterministic / stochastic description
  - => difficult to know what really counts...

Very few data is available on the experimental side and most of the time it is time or spatially integrated....

## IGNITION SYSTEM STUDIES

Small scale / lab-scale configuration  
<1ms



- Plasma physics
- Ignition chemistry
- Heat transfer

Input for a model  
initial state

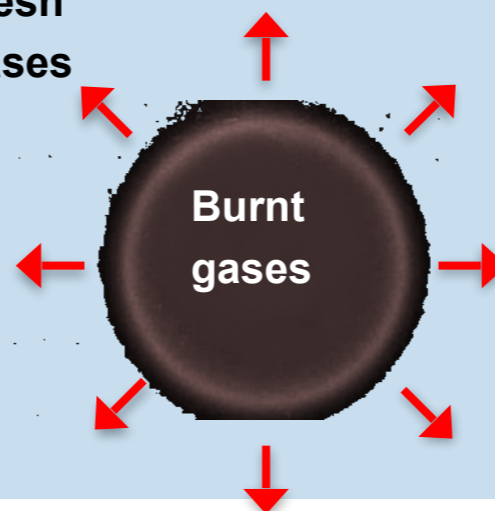


## IGNITION TRANSIENT STUDIES (NUMERICS / LES)

### Effect of stretch on the fuel consumption speed

$$S_c = S_L^0 - \mathcal{L}_a^c \kappa \quad \text{with} \quad \kappa = \frac{1}{S} \frac{\partial S}{\partial t} \quad (\text{stretch})$$

Fresh gases

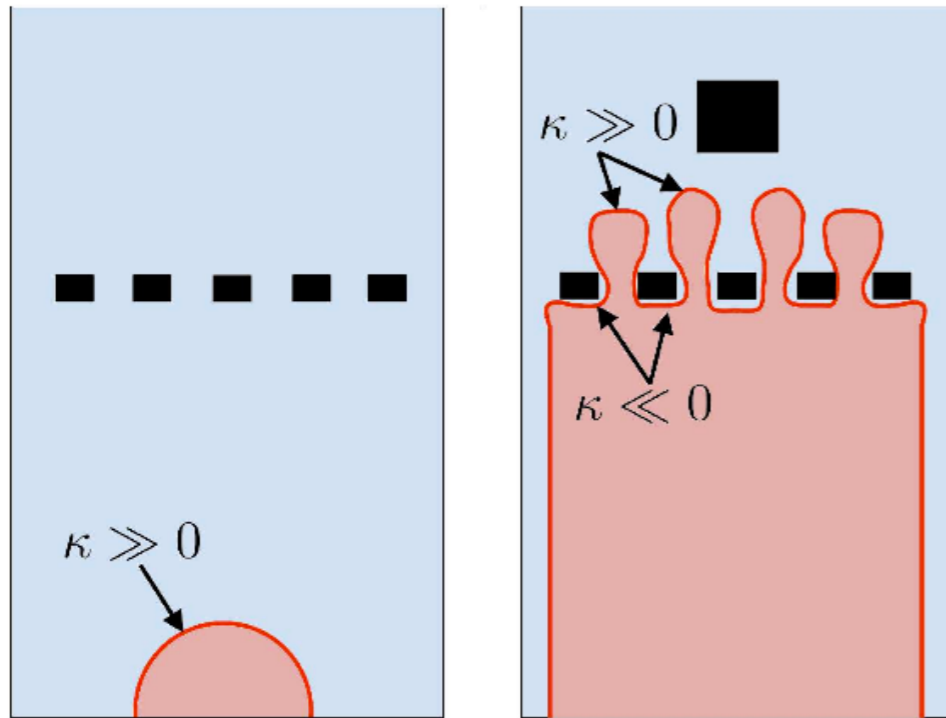


$$\mathcal{L}_a^c = \frac{1}{2} \beta \frac{T_{FG}}{T_{BG} - T_{FG}} \delta_L^0$$

$$\int_0^{\frac{T_{BG} - T_{FG}}{T_{FG}}} \frac{\ln(1+x)}{x} dx$$

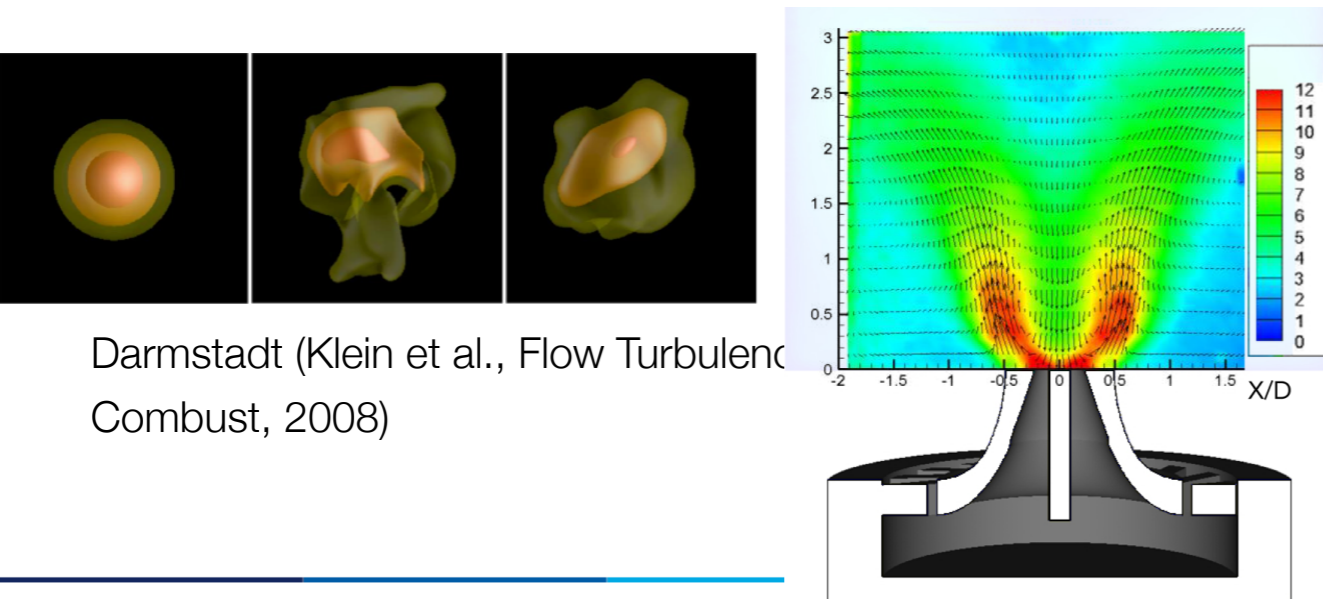


## 1/ Potential importance in deflagration simulations:



- Stretch definitely can play a role in the early Instants (always spheric and laminar)
- Thermo diffusive instabilities can appear here !!  
⇒ Impact on the local transition to turbulence (?)
- Stretch is also present when the flame front reaches obstacles...

## 2/ Potential importance in GT simulations:



Darmstadt (Klein et al., Flow Turbulence Combust, 2008)

- Stretch is rapidly present and strongly impact the initial kernel behavior (quenching...)
- As the flame propagates in the turbulent flow, it faces very different turbulent flow states...



# Potential chemistry modeling issues

## Thermo-chemical model and its impact

Flame propagation: C3H8, 0 array

$\phi = 1$	$S_L^0$	$T_{ad}$
------------	---------	----------

GRI-MECH 38.4 cm/s 2275 K

2-steps 38.4 cm/s 2289 K

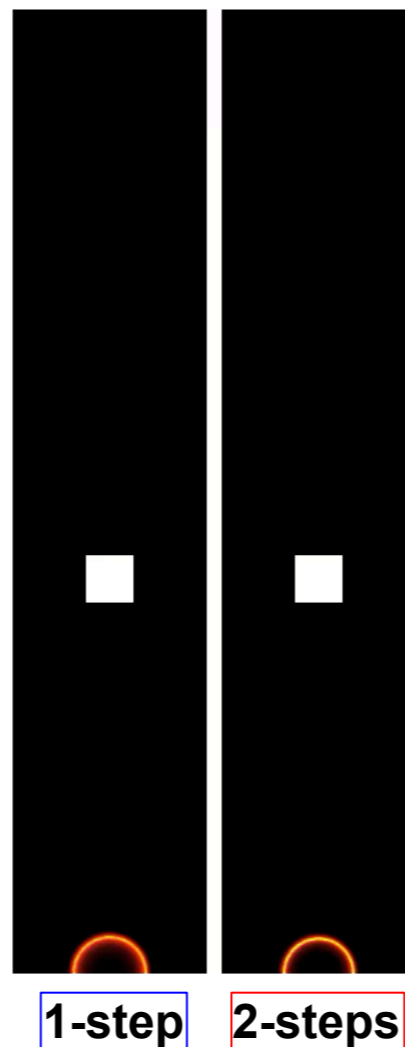
1-step 38.4 cm/s 2400 K

- $\phi = 1$  : same laminar flame speed
- 1-step adiab. Temp. overestimated by 5%

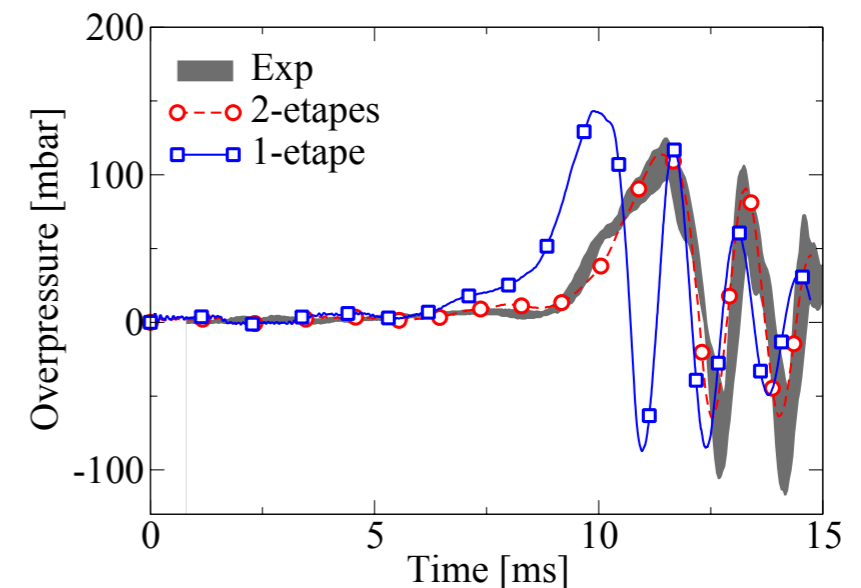
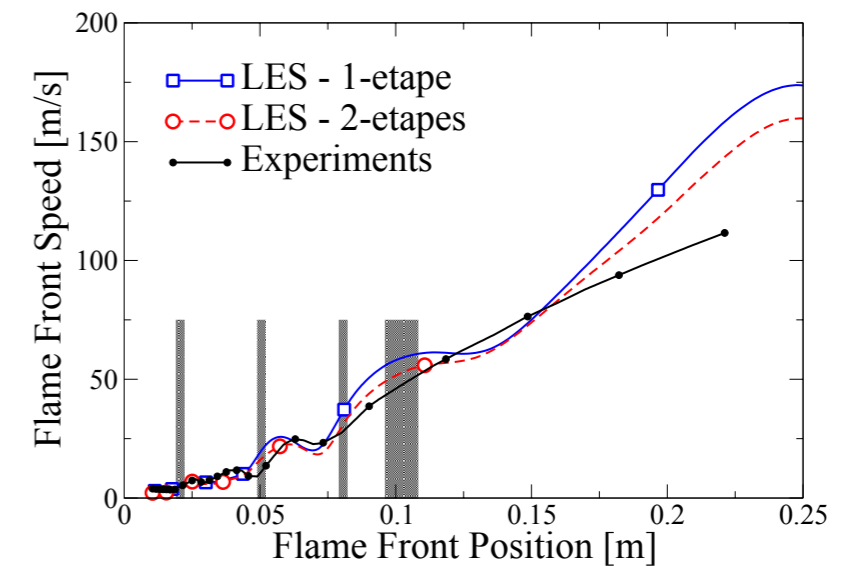
$$\Rightarrow S_d = \rho_{GF}/\rho_{GB} * S_L^0$$

overshot

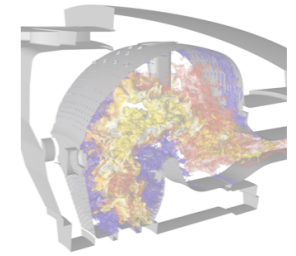
Time : 0.1 ms



Characteristic diag.: C3H8, 3-arrays



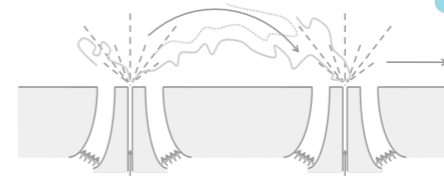
## I ] HPC & turbulent reacting flow CFD



## II ] Ignition / transient turbulent reacting flows

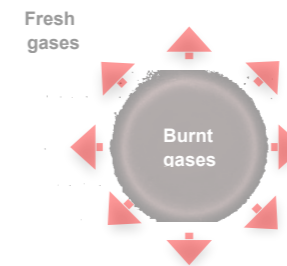
2.1 GT context: engine ignition prediction

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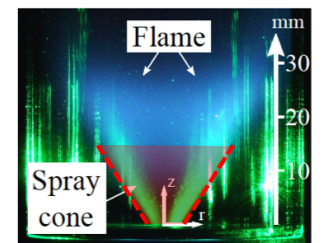


Buncefield, 2005

## III ] Difficulty of the initial phase



## IV ] GT applications: emission predictions & multi phase flows



## Conclusions and perspectives

# IV ] Application to a GT: pollutant emissions & multi-phase flows

Today reduced chemistry schemes are accessible in terms of tools and CPU<sup>4-6</sup>

## Analytically Reduced Chemistry (ARC) for C<sub>3</sub>H<sub>8</sub> / air combustion

**Pepiot-Desjardins / Jerzembeck skeletal scheme<sup>1</sup>:**

Derived from the LLNL comprehensive mechanisms  
99 transported species  
669 reactions

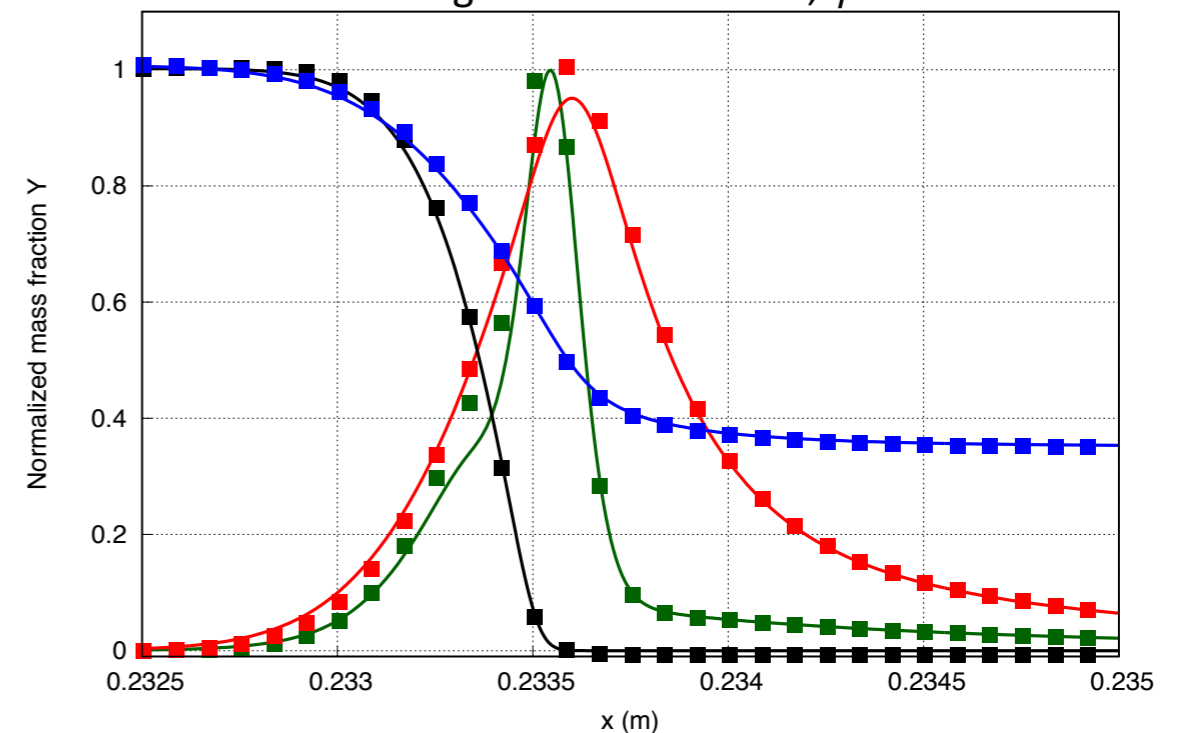
reference mechanism => skeletal mechanism:  
DRGEP<sup>2</sup>

skeletal mechanism => reduced scheme:  
QSSA using the LOI<sup>3</sup> criterion

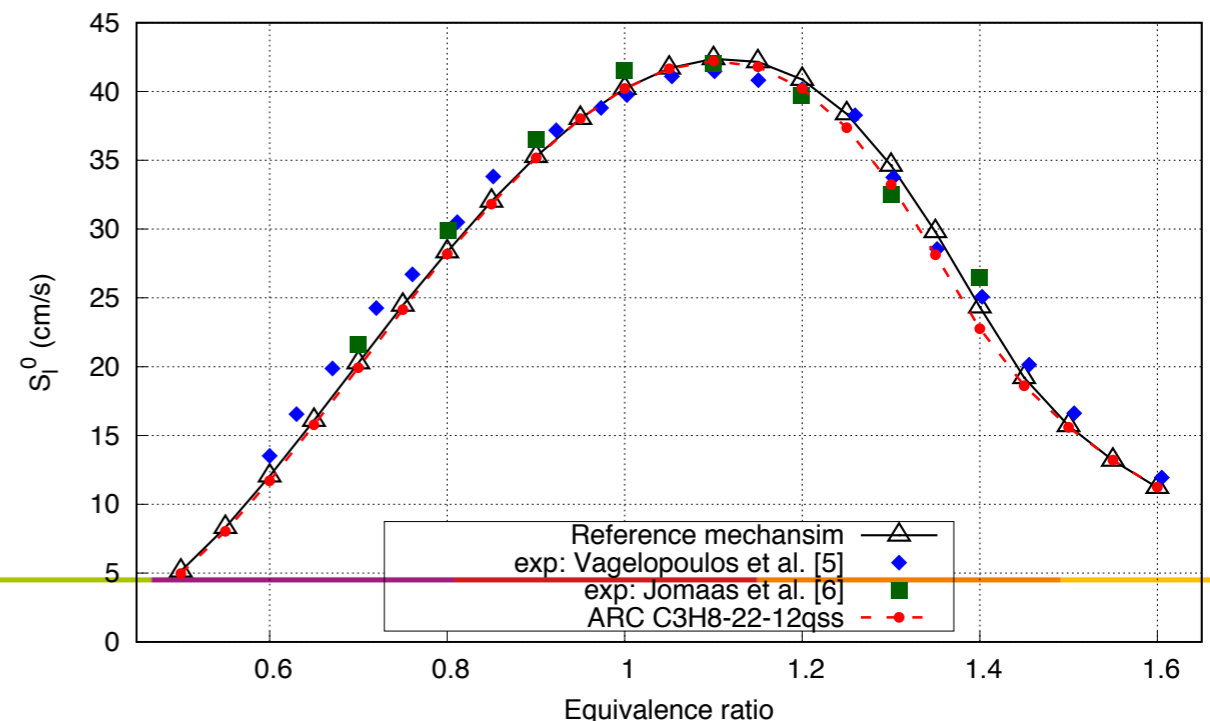
**ARC C<sub>3</sub>H<sub>8</sub>-22-12qss:**

22 transported species  
173 reactions  
12 QSS species

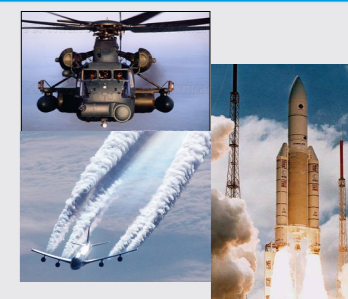
1D flame using AVBP at T = 288 K,  $\phi = 0.65$



$S_1^0$  for T= 288 K as function of the equivalence ratio



<sup>1</sup> Jerzembeck, S., et al (2009). *Combustion and Flame*, 156(2), 292-301.  
<sup>2</sup> Pepiot-Desjardins, P., et al (2008). *Combustion and Flame*, 154(1), 67-81.  
<sup>3</sup> Løvås, T., et al (2002). *Proceedings of the Combustion Institute*, 29(1), 1387-1393.  
<sup>4</sup> Jaravel, T. et al (2016), *Proceedings of the Combustion Institute*, 36.  
<sup>5</sup> Schulz, O. et al (2016), *Proceedings of the Combustion Institute*, 36.  
<sup>6</sup> Felden, A. et al (2016), *Proceedings of the Combustion Institute*, 36.



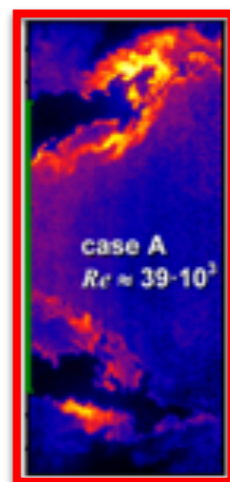
*OH PLIF [I]*

*LES*

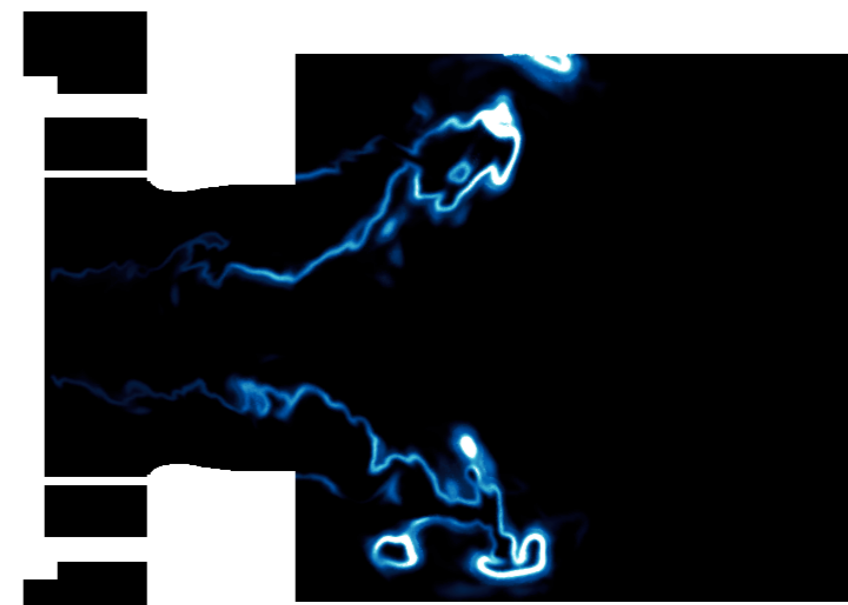
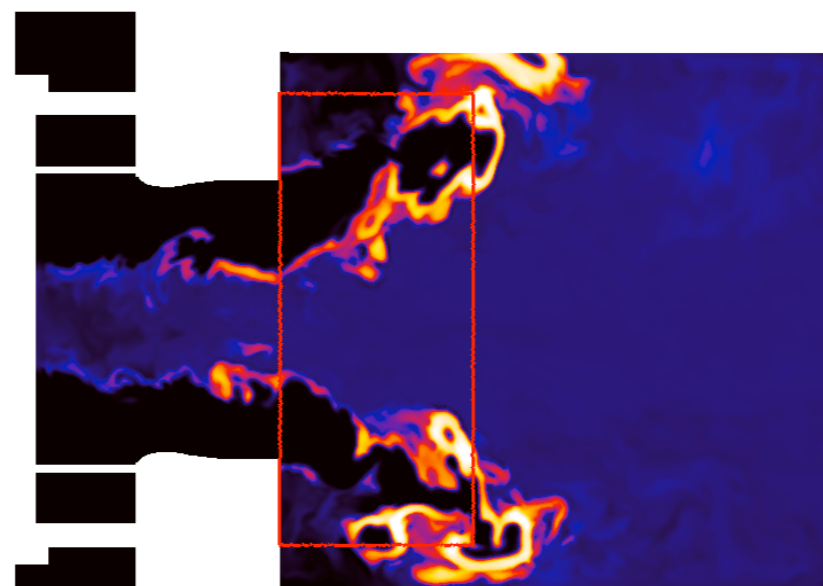
*OH Concentration*

*Heat release rate*

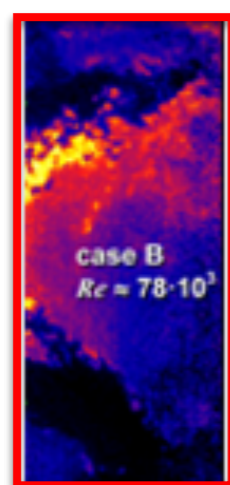
*3 Bars*



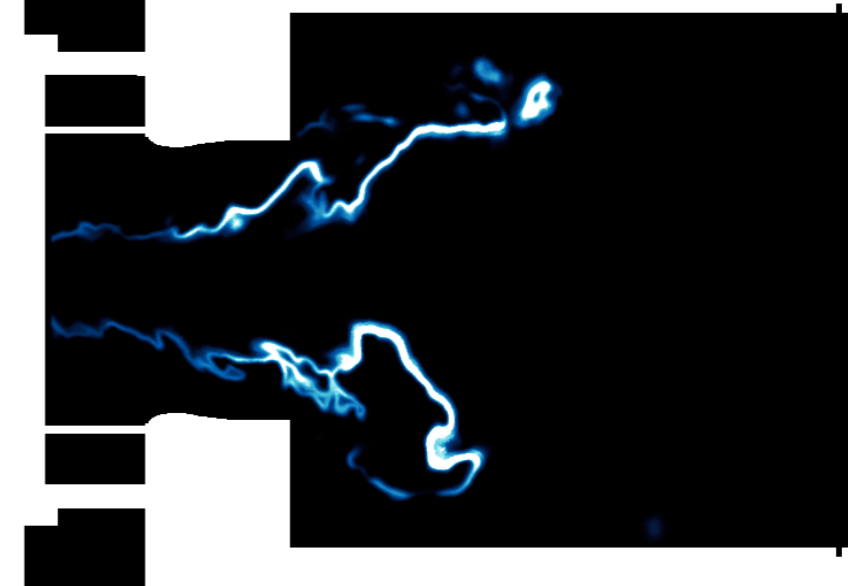
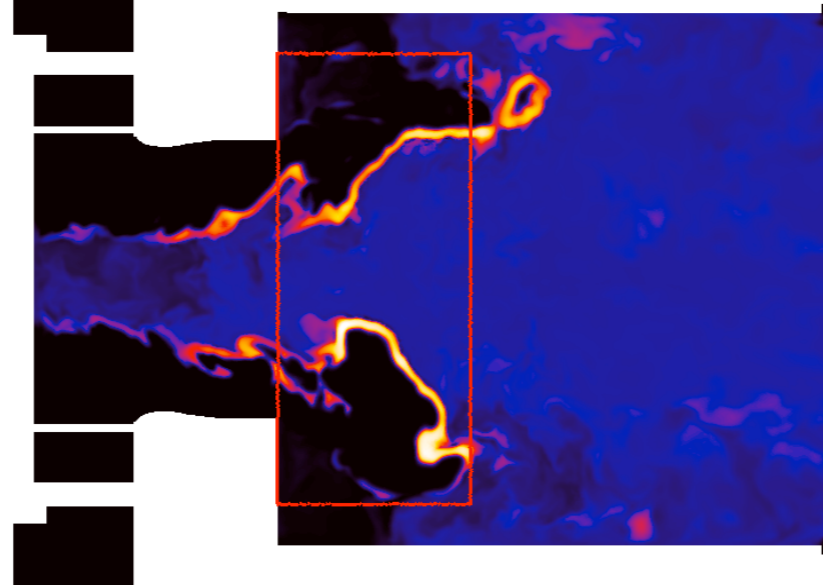
case A  
 $Re \approx 39 \cdot 10^3$



*6 Bars*



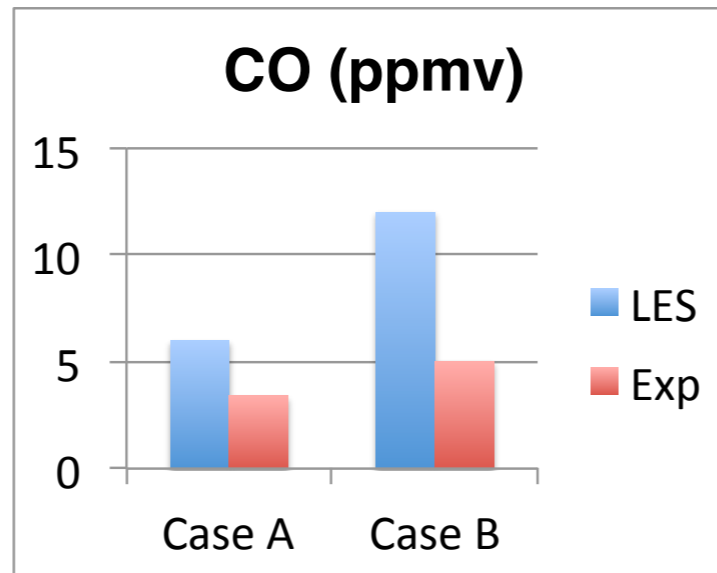
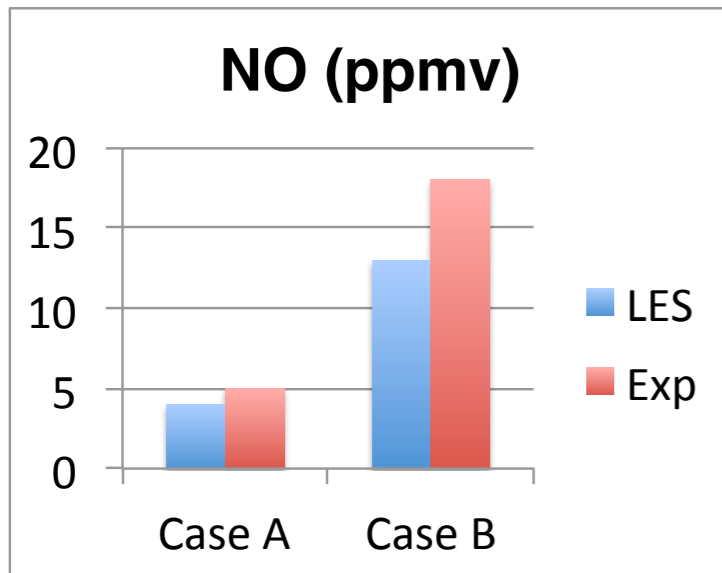
case B  
 $Re \approx 78 \cdot 10^3$



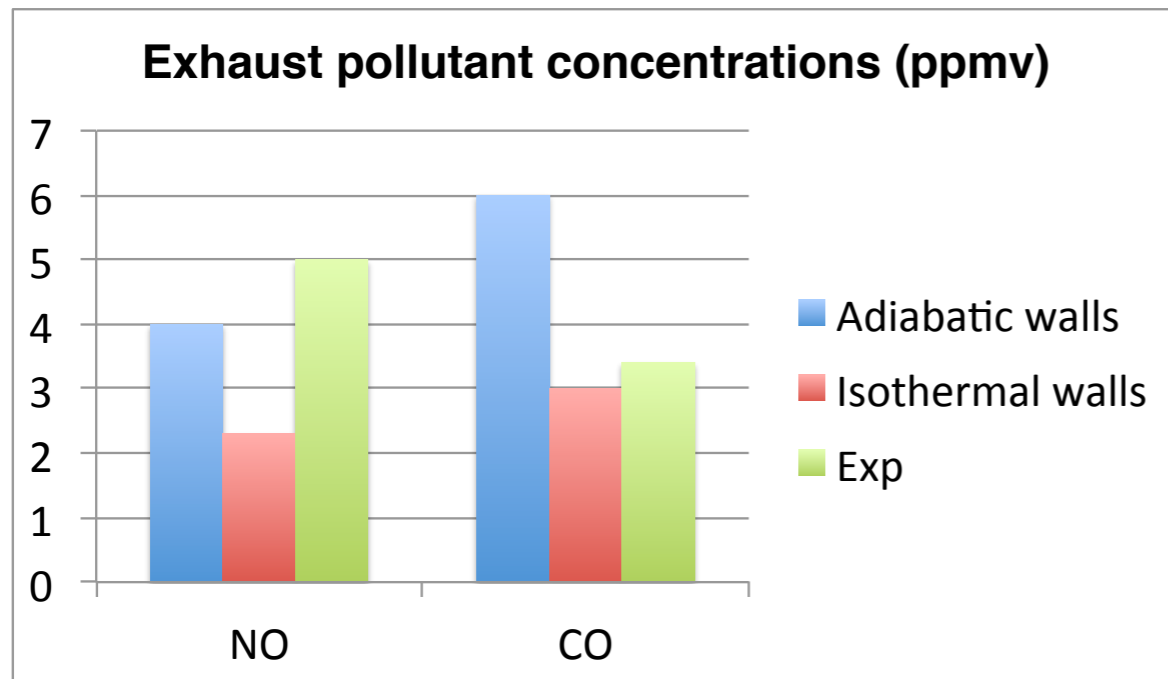
0.0 70.0 140.0 210.0 300.0

$HRR/\rho$  [W/kg]

## Exhaust pollutant concentrations



- NO
  - Satisfactory prediction
  - Slight under-prediction
  - Trend correctly recovered
- CO
  - Significant over-prediction

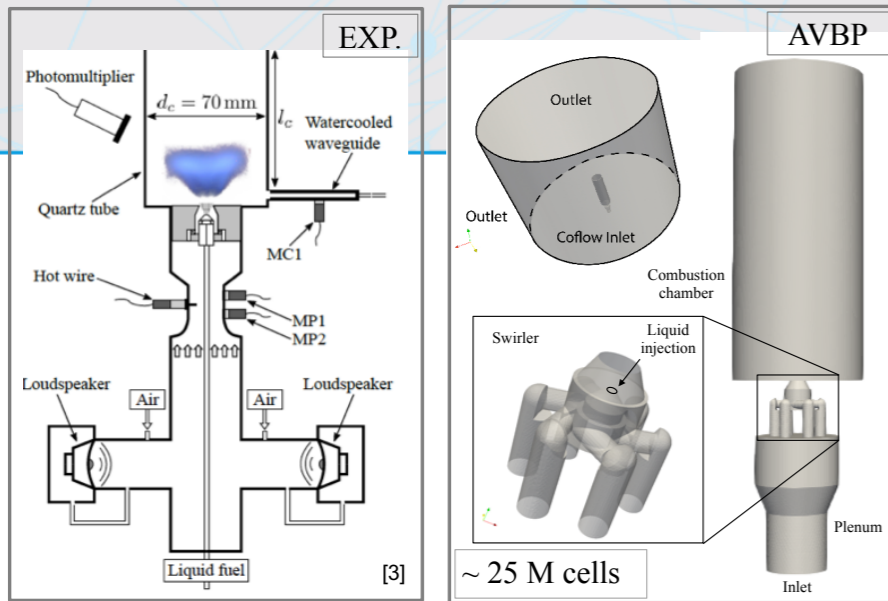


Deterioration of **NO prediction**

➔ Improvement of **temperature prediction**

➔ Significant improvement of **CO prediction** at the exit: driven by **equilibrium**

# Single burner setup: SICCA-Spray

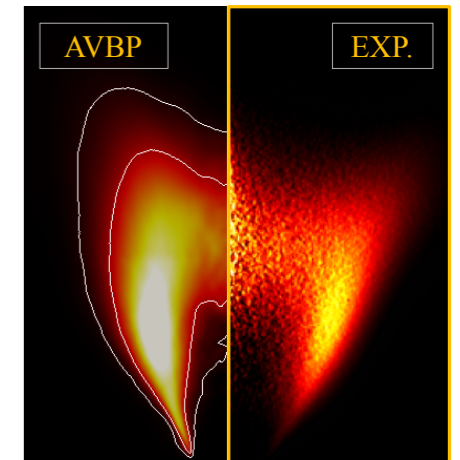
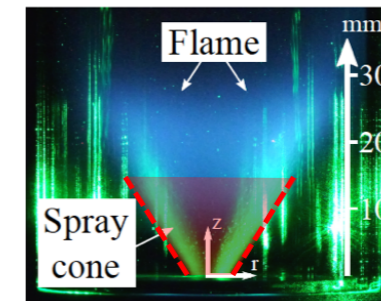
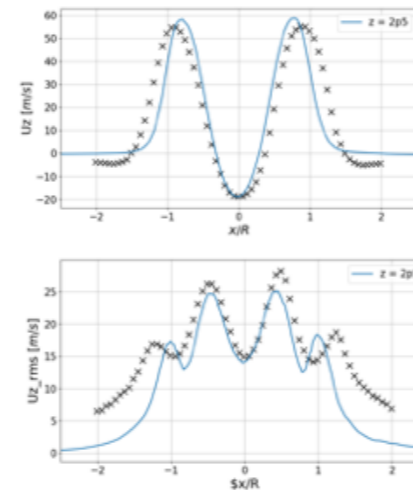


EM2C laboratory (Paris)

# Multi-phase reacting flows

## Steady operating conditions

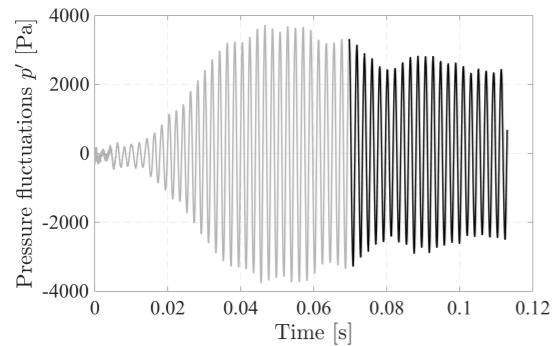
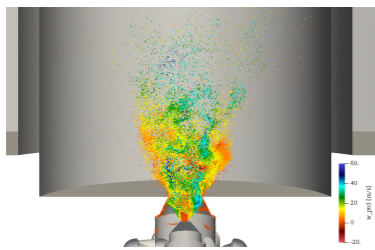
- cold non-reacting
- reacting spray flame



## Thermo acoustically unstable configuration

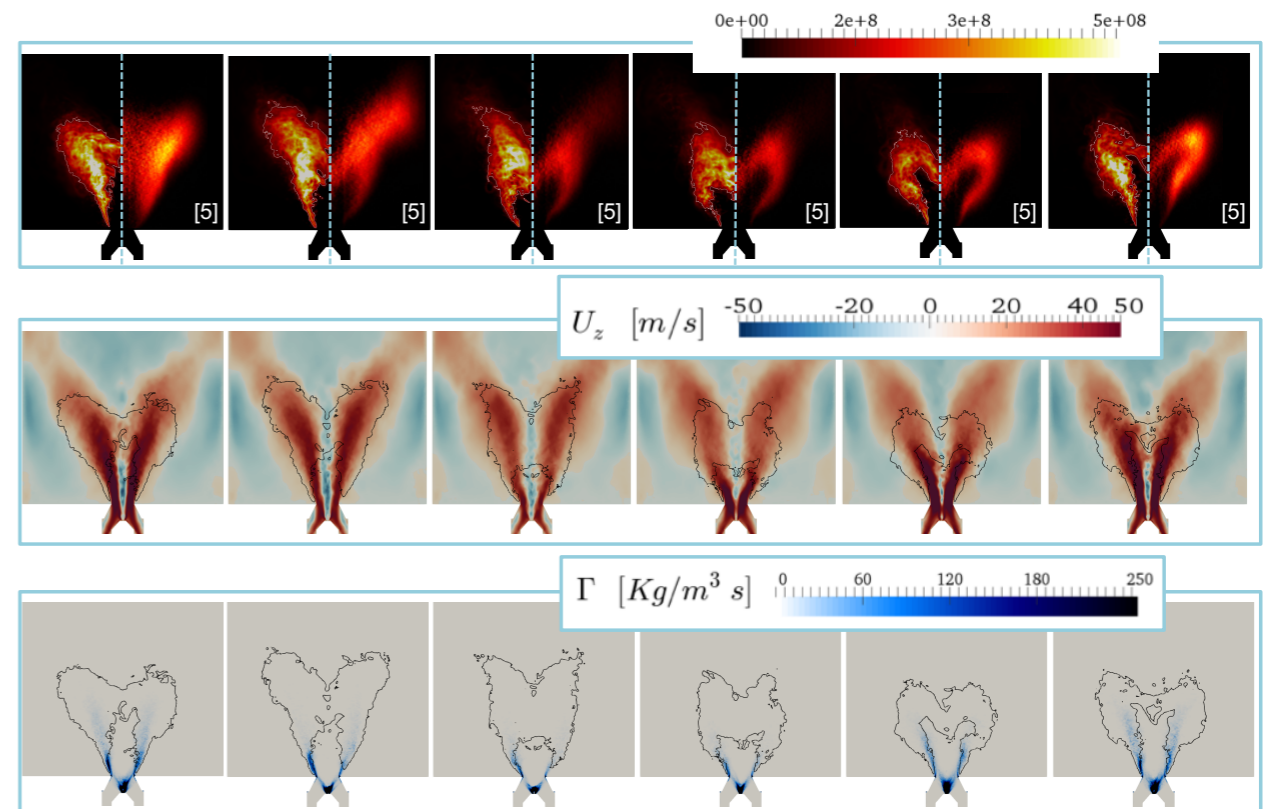
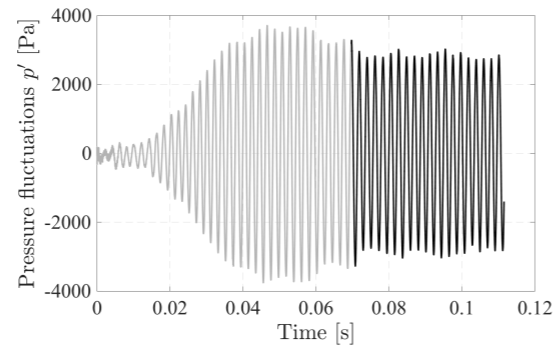
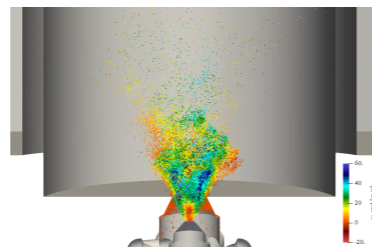
Wall interaction model:

**SLIP**



Wall interaction model:

**FILM**



# Conclusions & perspectives

## **Current progresses in LES @ CERFACS include more and more complexity**

=> In terms of chemistry: ARC schemes needed to predict pollutants will improve the quality of the laminar flame speed predictions for simple flames provided that these schemes are properly constructed from adequate reference schemes.

=> ARC will however not alleviate the dependency of the turbulent combustion closure to the laminar flame speed (and thickness) for non-planar flames... i.e. how to properly incorporate stretch and strain effects

=> Multi-phase flames clearly add complexity: depending on the droplets, different regimes of combustion appear and their effect on the flame thickness and speed are not fully understood...